

Table 3 Suction, Discharge, and Liquid Line Capacities in Tons for Refrigerant 22 (Single- or High-Stage Applications)

Line Size		Suction Lines ($\Delta t = 2^{\circ}\text{F}$)					Discharge Lines ($\Delta t = 1^{\circ}\text{F}$, $\Delta p = 3.05\text{ psi}$)		Line Size	Liquid Lines		
		Saturated Suction Temperature, $^{\circ}\text{F}$					Saturated Suction Temperature, $^{\circ}\text{F}$		Type L, Copper, OD	See notes a and b		
		-40	-20	0	20	40	-40	40		Vel., = 100 fpm	$\Delta t = 1^{\circ}\text{F}$ $\Delta p = 3.05$	
Type L, Copper, OD		0.79	1.15	1.6	2.22	2.91						
1/2	—	—	—	—	0.40	0.6	0.75	0.85	1/2	2.5	3.6	
5/8	—	—	0.32	0.51	0.76	1.1	1.4	1.6	5/8	3.7	6.7	
7/8	0.52	0.86	1.3	2.0	2.9	—	3.7	4.2	7/8	7.6	18.2	
1 1/8	1.1	1.7	2.7	4.0	5.8	—	7.5	8.5	1 1/8	13.2	37.0	
1 3/8	1.9	3.1	4.7	7.0	10.1	—	13.1	14.8	1 3/8	20.2	64.7	
1 5/8	3.0	4.8	7.4	11.1	16.0	—	20.7	23.4	1 5/8	28.5	102.5	
2 1/8	4.2	6.6	10.0	15.1	23.1	—	32.6	39.5	2 1/8	49.6	213.0	
2 5/8	10.9	17.8	27.5	40.8	58.3	—	73.4	85.4	2 5/8	76.5	376.9	
3 1/8	17.5	28.4	44.0	65.0	92.9	—	120.2	136.2	3 1/8	109.2	601.5	
3 5/8	26.0	42.3	65.4	96.6	137.8	—	178.4	202.1	3 5/8	147.8	895.7	
4 1/8	36.8	59.6	92.2	136.3	194.3	—	253.1	284.4	4 1/8	192.1	1263.2	
Steel									Steel			
IPS	SCH 40								IPS	SCH 40		
1/2	40	—	0.58	0.58	0.85	1.2	1.5	1.7	1/2	800	3.8	5.7
3/4	40	0.50	0.8	1.2	1.8	2.5	3.3	3.7	3/4	800	6.9	12.8
1	40	0.95	1.5	2.3	3.4	4.8	6.1	6.9	1	800	11.5	25.2
1 1/4	40	2.0	3.2	4.8	7.0	9.9	12.6	14.3	1 1/4	800	20.6	54.1
1 1/2	40	3.0	4.7	7.2	10.5	14.8	19.0	21.5	1 1/2	800	28.3	82.6
2	40	5.7	9.1	13.9	20.2	28.5	36.8	41.4	2	400	53.8	192.0
2 1/2	40	9.2	14.6	22.1	32.7	45.4	58.1	65.9	2 1/2	400	78.7	305.8
3	40	14.2	23.7	36.0	54.8	80.1	102.8	116.4	3	400	118.5	540.3
4	40	23.1	37.5	56.8	85.9	123.2	159.5	177.3	4	400	204.2	1001.2

Notes:

1. Table capacities are in tons of refrigeration.

 Δp = pressure drop from line friction, psi per 100 ft of equivalent line length Δt = corresponding change in saturation temperature, $^{\circ}\text{F}$ per 100 ft2. Line capacity for other saturation temperatures Δt and equivalent lengths L_e

$$\text{Line capacity} = \text{Table capacity} \left(\frac{\text{Table } L_e}{\text{Actual } L_e} \right)^{1.8} \left(\frac{\text{Actual } \Delta t}{\text{Table } \Delta t} \right)^{1.8}$$

3. Saturation temperature Δt for other capacities and equivalent lengths L_e

$$\Delta t = \text{Table } \Delta t \left(\frac{\text{Actual } L_e}{\text{Table } L_e} \right)^{1.8} \left(\frac{\text{Actual capacity}}{\text{Table capacity}} \right)^{1.8}$$

*Sizing shown is recommended where any gas generated in receiver must return up condenser line to condenser without restricting condenser flow. Water-cooled condensers, where receiver ambient temperature may be higher than refrigerant condensing temperature, fall into this category.

4. Values based on 105°F condensing temperature. Multiply table capacities by the following factors for other condensing temperatures.

Condensing Temperature, $^{\circ}\text{F}$	Suction Line	Discharge Line
80	1.11	0.79
90	1.07	0.88
100	1.03	0.95
110	0.97	1.04
120	0.90	1.10
130	0.84	1.18
140	0.80	1.26

*Line pressure drop Δp is conservative; if subcooling is substantial or line is short, a smaller size line may be used. Applications with very little subcooling on very long lines may require a larger line.

Table 4 Suction, Discharge, and Liquid Line Capacities in Tons for Refrigerant 22 (Intermediate- or Low-Stage Duty)

Line Size		Suction Lines ($\Delta t = 2^{\circ}\text{F}$) ^a					Discharge Lines ($\Delta t = 2^{\circ}\text{F}$) ^a		Liquid Lines	
		Saturated Suction Temperature, $^{\circ}\text{F}$								
		-90	-80	-70	-60	-50	-40	-30		
5/8	—	—	—	—	—	—	—	—		
7/8	0.18	0.25	0.34	0.46	0.61	0.79	1.0	1.9		
1 1/8	0.34	0.51	0.70	0.94	1.2	1.6	2.1	3.8		
1 3/8	0.6	0.9	1.2	1.6	2.2	2.8	3.6	6.6		
1 5/8	1.0	1.4	1.9	2.6	3.4	4.3	5.7	10.5		
2 1/8	2.1	3.0	4.1	5.5	7.2	9.3	11.9	21.7		
2 5/8	3.8	5.3	7.2	9.7	12.7	16.5	21.1	38.4		
3 1/8	6.1	8.5	11.6	15.5	20.4	26.4	33.8	61.4		
3 5/8	9.1	12.7	17.3	23.1	30.4	39.4	50.2	91.2		
4 1/8	12.9	18.0	24.5	32.7	43.0	55.6	70.9	128.6		
5 1/8	23.2	32.3	43.9	58.7	77.1	99.8	126.9	229.5		
6 1/8	37.5	52.1	71.0	94.6	124.2	160.5	204.2	369.4		

Notes:

1. Table capacities are in tons of refrigeration.

 Δp = pressure drop from line friction, psi per 100 ft of equivalent line length Δt = corresponding change in saturation temperature, $^{\circ}\text{F}$ per 100 ft2. Line capacity for other saturation temperatures Δt and equivalent lengths L_e

$$\text{Line capacity} = \text{Table capacity} \left(\frac{\text{Table } L_e}{\text{Actual } L_e} \right)^{1.8} \left(\frac{\text{Actual } \Delta t}{\text{Table } \Delta t} \right)^{1.8}$$

3. Saturation temperature Δt for other capacities and equivalent lengths L_e

$$\Delta t = \text{Table } \Delta t \left(\frac{\text{Actual } L_e}{\text{Table } L_e} \right)^{1.8} \left(\frac{\text{Actual capacity}}{\text{Table capacity}} \right)^{1.8}$$

4. Refer to refrigerant thermodynamic property tables (Chapter 20 of the 2013 ASHRAE Handbook—Fundamentals) for pressure drop corresponding to Δt .

^aSee section on Pressure Drop Considerations.

5. Values based on 0°F condensing temperature. Multiply table capacities by the following factors for other condensing temperatures. Flow rates for discharge lines are based on -50°F evaporating temperature.

Condensing Temperature, $^{\circ}\text{F}$	Suction Line	Discharge Line
-30	1.09	0.58
-20	1.06	0.71
-10	1.03	0.83
0	1.00	1.00
10	0.97	1.20
20	0.94	1.43
30	0.90	1.80

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James Ray Jones



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Ventilation and Infiltration American Society of Heating, Refrigerating and Air-Conditioning Engineers, 2001

Handbook of Climate Change Mitigation and Adaptation Maximilian Lackner, Baharak Sajjadi, Wei-Yin Chen, 2025-09-26

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Building Ventilation Mat Santamouris, Peter Wouters, 2006-06-01 Ensuring optimum ventilation performance is a vital part of building design Prepared by recognized experts from Europe and the US and published in association with the International Energy Agency's Air Infiltration and Ventilation Centre AIVC this authoritative work provides organized classified and evaluated information on advances in the key areas of building ventilation relevant to all building types Complexities in airflow behaviour climatic influences occupancy patterns and pollutant emission characteristics make selecting the most appropriate ventilation strategy especially difficult Recognizing such complexities the editors bring together expertise on each key issue From components to computer tools this book offers detailed coverage on design analysis and performance and is an important and comprehensive publication in this field Building Ventilation will be an invaluable reference for professionals in the building services industry architects researchers including postgraduate students studying building service engineering and HVAC and anyone with a role in energy efficient building design

Masonry Walls for Energy Conservation James E. Amrhein, 1981 *Occurrence of Mold in a Two-story Wood-frame House Operated at Design Indoor Humidity Levels*, 2009 Mold growth was observed in limited areas in a two story contemporary wood frame house during the seventh heating season of operation at or near design indoor humidity levels The house was in a cold climate Madison Wisconsin Humidity levels were estimated as being exceeded in one of 10 homes at this location Moderate amounts of window condensation were observed during each of the seven heating seasons Areas of observed mold growth were limited to the lower extremities of windows near the lower edge of glass panes and the edges of lower panels of a wood panel entry door that was not equipped with a storm door Mold growth was also present in sprayed on cellulose insulation in close proximity to the rim joist board in the basement although this mold was not visible unless the insulation was disturbed and was probably present during previous heating seasons Mold from these locations was cultured isolated and identified morphologically or by DNA sequence Seven genera of common ascomycetes and deuteromycetes were detected all of which are commonly associated with indoor air *Penicillium* was the most common genus with at least six different species The greatest variety of genera five occurred in samples taken from interior wood millwork the edges of panels in the entry door and from a wood window sash Only two genera were found in a sample taken from the interface between the rim joist board and cellulose insulation Total spore counts taken in April revealed that on the first and second stories mold spores were less prevalent than in outdoor air but that mold spores were more prevalent in the basement than in outdoor air

[Solutions Manual for the Mechanical Engineering Reference Manual](#) Michael R. Lindeburg, 1994

Fundamentals of Building Energy Dynamics Bruce D. Hunn, 1996 Fundamentals of Building Energy Dynamics assesses how and why buildings use energy and how energy use and peak demand can be reduced It provides a basis for integrating energy efficiency and solar approaches in ways that will allow building owners and designers to balance the need to minimize initial costs operating costs and life cycle costs with

need to maintain reliable building operations and enhance environmental quality both inside and outside the building. Chapters trace the development of building energy systems and analyze the demand side of solar applications as a means for determining what portion of a building's energy requirements can potentially be met by solar energy. Following the introduction, the book provides an overview of energy use patterns in the aggregate U.S. building population. Chapter 3 surveys work on the energy flows in an individual building and shows how these flows interact to influence overall energy use. Chapter 4 presents the analytical methods, techniques, and tools developed to calculate and analyze energy use in buildings, while chapter 5 provides an extensive survey of the energy conservation and management strategies developed in the post energy crisis period. The approach taken is a commonsensical one starting with the proposition that the purpose of buildings is to house human activities and that conservation measures that negatively affect such activities are based on false economies. The goal is to determine rational strategies for the design of new buildings and the retrofit of existing buildings to bring them up to modern standards of energy use. The energy flows examined are both large scale heating systems and small scale choices among appliances.

Solar Heat Technologies: Fundamentals and Applications, Volume 4 **Plant Engineer's Handbook of Formulas, Charts, and Tables** Donald W. Moffat, 1982 **Architecture** United States. Naval Facilities Engineering Command, 1984 Research Paper FPL-RP, 1986 **Building Energy Efficiency Standards** California Energy Commission, 1986 **Ramsey/Sleeper architectural graphic standards** Charles George Ramsey, Harold Reeve Sleeper, John Ray Hoke, American Institute of Architects, 1994-03-21. The architect's standard reference for over 60 years is now available as a book/CD-ROM set. Use the book for ideas and inspiration. Then use one of five powerful search methods on the CD-ROM to quickly find all the information you need for a design project. Architectural Graphic Standards Charles George Ramsey, Harold Reeve Sleeper, 2000. With new features such as expanded design guidelines and new information on international building standards, this tenth edition will keep architects and design professionals up to date with over 50 percent new material from the previous edition. Illustrations.

Affordable Housing Through Energy Conservation, 1989 **Proceedings of the Summer Computer Simulation Conference**, 1982 **Predictive Equations for Nighttime Ventilation Control Based on Building Mass Temperature** James Ray Jones, 1996 National Mechanical Estimator Victor B. Ottaviano, 1993 Handbook of Mechanical and Electrical Systems for Buildings Harry E. Bovay, 1981

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