

Table 3 Section, Discharge, and Liquid Line Capacities in Tons for Refrigerant 22 (Single- or High-Stage Applications)

Line Size	Section Lines ( $\Delta t = 2^\circ\text{F}$ )					Discharge Lines ( $\Delta t = 1^\circ\text{F}$ , $\Delta p = 3,000\text{ psig}$ )	Line Size	Liquid Lines			
	Saturated Section Temperatures, $^{\circ}\text{F}$							Saturated			
	-400	-200	0	200	400	-400	-200	-200	0	200	
Type L Copper, OD	0.79	1.15	1.46	2.22	2.98	-400	-400	0.79	1.15	1.46	
1/2	—	—	—	—	—	—	—	1/2	—	—	
5/8	—	—	—	—	—	—	—	5/8	—	—	
7/8	0.52	0.56	1.3	2.0	2.4	—	—	7/8	—	—	
1 1/8	1.1	1.7	2.7	4.0	5.8	—	—	1 1/8	—	—	
1 3/8	1.9	3.1	4.7	7.0	10.1	1 3/8	1 3/8	1 3/8	2 1/8	—	
1 5/8	3.0	4.8	7.9	11.1	16.0	2 1/8	2 1/8	2 1/8	2 1/8	—	
2 1/8	4.2	6.6	10.6	13.8	18.8	4 1/8	4 1/8	4 1/8	4 1/8	—	
2 5/8	6.9	11.3	17.3	20.8	26.3	7 1/8	7 1/8	7 1/8	7 1/8	—	
3 1/8	9.3	13.4	24.0	26.9	32.9	10 1/8	10 1/8	10 1/8	10 1/8	—	
3 5/8	24.0	42.3	67.6	70.6	137.8	12 1/8	12 1/8	12 1/8	12 1/8	—	
4 1/8	36.8	59.6	92.7	125.3	194.8	22 1/8	22 1/8	22 1/8	22 1/8	—	
<b>Table 1</b>											
<b>Table 2</b>											
1/2	400	—	60.50	60.50	60.50	1/2	1/2	60.50	60.50	60.50	
3/8	400	60.50	1.2	1.8	2.5	3/8	3/8	60.50	60.50	60.50	
1	400	60.50	2.5	3.4	4.8	1	1	60.50	60.50	60.50	
1 1/8	400	2.9	3.2	4.8	7.0	1 1/8	1 1/8	60.50	60.50	60.50	
1 3/8	400	3.0	4.7	7.2	10.8	1 3/8	1 3/8	60.50	60.50	60.50	
2	400	5.7	9.1	13.9	20.2	2	2	60.50	60.50	60.50	
2 1/8	400	9.2	14.6	22.1	32.2	4 1/8	4 1/8	60.50	60.50	60.50	
3	400	16.2	25.7	39.0	55.8	6 1/8	6 1/8	60.50	60.50	60.50	
4	400	33.8	52.5	79.0	115.9	10 1/8	10 1/8	60.50	60.50	60.50	

Notes:

1. Table capacities are in tons of refrigeration.

2. Gross capacity along these line sections, per 1000 ft of equivalent line length.  $\Delta t$  = corresponding change in saturation temperature,  $^{\circ}\text{F}$  (per 1000 ft).3. Line capacity for other saturation temperatures  $\Delta t$  and equivalent lengths  $L_e$ .

$$\text{Line capacity} = \text{Table capacity} \left[ \frac{\text{Table } L_e}{\text{Actual } L_e} \cdot \frac{\text{Actual } \Delta t}{\text{Table } \Delta t} \right]$$

4. Saturation temperatures  $\Delta t$  for other capacities and equivalent lengths  $L_e$ .

$$\Delta t = \text{Table } \Delta t \left[ \frac{\text{Actual } L_e}{\text{Table } L_e} \cdot \frac{\text{Actual capacity}}{\text{Table capacity}} \right]$$

5. Sizing factors are recommended where net gas generation in receiver vessel reduces equivalent line to condenser without reducing condenser flow. Water-cooled condensers, where receiver ambient temperature may be higher than refrigerant condensing temperature, fall into this category.

6. Values based on  $100^\circ\text{F}$  condensing temperature. Multiply table capacities by the following factors for other condensing temperatures.7. Condensing Temperature,  $^{\circ}\text{F}$  Section Line Discharge Line

Condensing Temperature, $^{\circ}\text{F}$	Section Line	Discharge Line
-400	1.2	1.2
-200	1.2	0.9
0	1.2	0.7
200	1.2	0.5
400	1.2	0.4
600	1.2	0.3
800	1.2	0.2
1000	1.2	0.1
1200	1.2	0.08
1400	1.2	0.06
1600	1.2	0.05
1800	1.2	0.04
2000	1.2	0.03
2200	1.2	0.02
2400	1.2	0.01

8. Line pressure drop  $\Delta p$  to overcome effects of subcooling or sublimation or lines in short, or smaller size lines may be used. Applications with very little subcooling or very long lines may require a longer line.

Table 4 Section, Discharge, and Liquid Line Capacities in Tons for Refrigerant 22 (Intermediate- or Low-Stage Duty)

Line Size	Section Lines ( $\Delta t = 2^\circ\text{F}$ )							Discharge Lines ( $\Delta t = 2^\circ\text{F}$ )	Liquid Lines		
	Saturated Section Temperatures, $^{\circ}\text{F}$										
	-400	-200	0	200	-400	-200	0				
Type L Copper, OD	0.79	1.15	1.46	2.22	2.98	-400	-400	0.79	1.15	1.46	
5/8	—	—	—	—	—	—	—	5/8	—	—	
7/8	0.52	0.56	1.3	2.0	2.4	—	—	7/8	—	—	
1 1/8	0.50	0.51	0.79	10.94	11.2	1.6	2.1	1 1/8	—	—	
1 3/8	0.6	1.2	2.6	2.8	2.8	3.6	4.6	1 3/8	—	—	
1 5/8	1.0	1.4	1.9	2.6	3.8	4.2	5.7	1 5/8	—	—	
2 1/8	2.1	3.0	4.2	5.5	7.2	9.3	11.8	2 1/8	—	—	
2 5/8	3.8	5.3	7.2	9.7	12.7	16.5	21.8	2 5/8	—	—	
3 1/8	6.1	8.5	11.6	13.5	20.8	26.4	33.8	3 1/8	—	—	
3 5/8	9.1	12.7	17.3	21.1	30.4	39.4	50.2	3 5/8	—	—	
4 1/8	12.9	18.0	24.5	32.7	43.0	55.6	70.8	4 1/8	—	—	
4 5/8	20.2	32.3	40.9	50.7	77.1	99.8	126.9	4 5/8	—	—	
5 1/8	37.9	52.1	77.0	94.6	124.2	160.9	204.2	5 1/8	—	—	

Notes:

1. Table capacities are in tons of refrigeration.

2. Gross capacity along these line sections, per 1000 ft of equivalent line length.  $\Delta t$  = corresponding change in saturation temperature,  $^{\circ}\text{F}$  (per 1000 ft).3. Line capacity for other saturation temperatures  $\Delta t$  and equivalent lengths  $L_e$ .

$$\text{Line capacity} = \text{Table capacity} \left[ \frac{\text{Table } L_e}{\text{Actual } L_e} \cdot \frac{\text{Actual } \Delta t}{\text{Table } \Delta t} \right]$$

4. Saturation temperatures  $\Delta t$  for other capacities and equivalent lengths  $L_e$ .

$$\Delta t = \text{Table } \Delta t \left[ \frac{\text{Actual } L_e}{\text{Table } L_e} \cdot \frac{\text{Actual capacity}}{\text{Table capacity}} \right]$$

5. Sizing factors are refrigerant thermodynamic property tables (Chapter 20 of the 2003 ASHRAE Handbook—Fundamentals) for pressure drop corresponding to  $\Delta t$ .

6. See section on Pressure Drop Considerations.

7. Values based on  $100^\circ\text{F}$  condensing temperature. Multiply table capacities by the following factors for other condensing temperatures. These values for discharge lines are based on  $-50^\circ\text{F}$  evaporation temperature.

Condensing Temperature, $^{\circ}\text{F}$	Section Line	Discharge Line
-400	1.079	0.998
-200	1.066	0.771
0	1.053	0.603
200	1.040	0.497
400	1.030	0.409
600	1.023	0.345
800	1.018	0.295
1000	1.014	0.256
1200	1.011	0.227
1400	1.008	0.201
1600	1.006	0.178
1800	1.004	0.157
2000	1.003	0.138
2200	1.002	0.121
2400	1.001	0.106

# Ashrae Chapter 26

**Victor B. Ottaviano**

## Ashrae Chapter 26:

**Ventilation and Infiltration** American Society of Heating, Refrigerating and Air-Conditioning Engineers, 2001

Handbook of Climate Change Mitigation and Adaptation Maximilian Lackner, Baharak Sajjadi, Wei-Yin Chen, 2025-09-26

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HVACR Principles and Applications Nuggenhalli S.

Nandagopal, 2024-03-07 This book provides a clear and concise understanding of the principles and applications of HVACR using a rigorous yet easy to follow presentation. The coverage is broad including relevant support areas such as fluid mechanics heat transfer thermodynamics psychrometrics with specific applications to HVACR design and calculations and main topics such as air conditioning processes cooling heating load calculations refrigeration cycles and HVACR equipment and systems. The book integrates and illustrates the use of data and information from ASHRAE Handbooks and Standards in step by step calculations of cooling and heating loads and other aspects of HVACR. Elucidation of the principles is further reinforced by examples and practice problems with detailed solutions. Firmly grounded in the fundamentals the book maximizes readers capacity to take on new problems and challenges in the field of HVACR with confidence and conviction. Providing a ready reference and review of essential principles and their applications in HVACR the book is ideal for HVACR practitioners undergraduate engineering students and those specializing in HVACR as well as for practicing engineers preparing for the engineering license exams FE and PE in USA and abroad. The book uses both Inch Pound I P and S I

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Amrhein, 1981      *Occurrence of Mold in a Two-story Wood-frame House Operated at Design Indoor Humidity Levels* , 2009 Mold growth was observed in limited areas in a two story contemporary wood frame house during the seventh heating season of operation at or near design indoor humidity levels The house was in a cold climate Madison Wisconsin Humidity levels were estimated as being exceeded in one of 10 homes at this location Moderate amounts of window condensation were observed during each of the seven heating seasons Areas of observed mold growth were limited to the lower extremities of windows near the lower edge of glass panes and the edges of lower panels of a wood panel entry door that was not equipped with a storm door Mold growth was also present in sprayed on cellulose insulation in close proximity to the rim joist board in the basement although this mold was not visible unless the insulation was disturbed and was probably present during previous heating seasons Mold from these locations was cultured isolated and identified morphologically or by DNA sequence Seven genera of common ascomycetes and deuteromycetes were detected all of which are commonly associated with indoor air Penicillium was the most common genus with at least six different species The greatest variety of genera five occurred in samples taken from interior wood millwork the edges of panels in the entry door and from a wood window sash Only two genera were found in a sample taken from the interface between the rim joist board and cellulose insulation Total spore counts taken in April revealed that on the first and second stories mold spores were less prevalent than in outdoor air but that mold spores were more prevalent in the basement than in outdoor air      Solutions Manual for the Mechanical Engineering Reference Manual Michael R. Lindeburg, 1994      **Fundamentals of Building Energy Dynamics** Bruce D. Hunn, 1996 Fundamentals of Building Energy Dynamics assesses how and why buildings use energy and how energy use and peak demand can be reduced It provides a basis for integrating energy efficiency and solar approaches in ways that will allow building owners and designers to balance the need to minimize initial costs operating costs and life cycle costs with

need to maintain reliable building operations and enhance environmental quality both inside and outside the building Chapters trace the development of building energy systems and analyze the demand side of solar applications as a means for determining what portion of a building's energy requirements can potentially be met by solar energy Following the introduction the book provides an overview of energy use patterns in the aggregate U.S. building population Chapter 3 surveys work on the energy flows in an individual building and shows how these flows interact to influence overall energy use Chapter 4 presents the analytical methods, techniques and tools developed to calculate and analyze energy use in buildings while chapter 5 provides an extensive survey of the energy conservation and management strategies developed in the post energy crisis period The approach taken is a commonsensical one starting with the proposition that the purpose of buildings is to house human activities and that conservation measures that negatively affect such activities are based on false economies The goal is to determine rational strategies for the design of new buildings and the retrofit of existing buildings to bring them up to modern standards of energy use The energy flows examined are both large scale heating systems and small scale choices among appliances Solar Heat Technologies Fundamentals and Applications Volume 4

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**Affordable Housing Through Energy Conservation: Technical support document**, 1989      **Proceedings of the Summer Computer Simulation Conference**, 1982      **Predictive Equations for Nighttime Ventilation Control Based on Building Mass Temperature** James Ray Jones, 1996      National Mechanical Estimator Victor B. Ottaviano, 1993

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